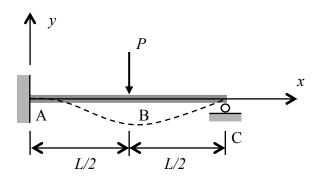
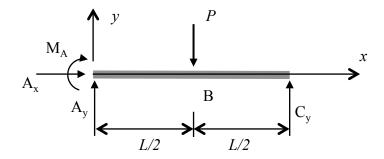
CE 160 Notes: Indeterminate Beam Example

For the indeterminate beam loaded with the point load, P, find the support reactions at A and B. EI is constant.



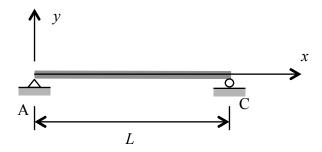
F.B.D neglecting axial effects



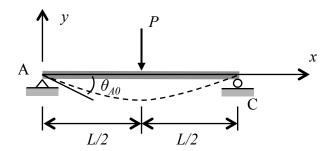
4 unknowns, 3 equations of equilibrium

Indeterminate to the First Degree

Define M_A as the redundant, so the released or primary structure is:



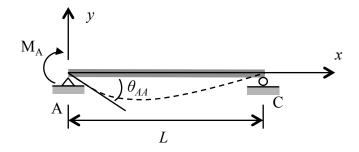
Released Structure Problem – Need to find the rotation at point A, $\theta_{A\theta}$

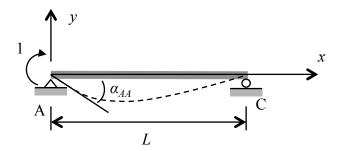


Use the solutions tabulated in the text Figure 11.3

$$\theta_{A0} = -\frac{PL^2}{16EI}$$

Redundant Problem – Need to find the flexibility coefficient, α_{AA}

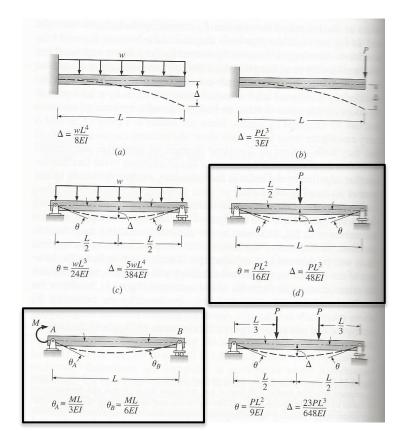




$$\theta_{AA}=M_A\alpha_{AA}$$

Use the solutions tabulated in the text Figure 11.3

$$\alpha_{AA} = -\frac{L}{3EI}$$



Compatibility Equation at Point A

$$\theta_{A0} + M_A \alpha_{AA} = \theta_A = 0$$

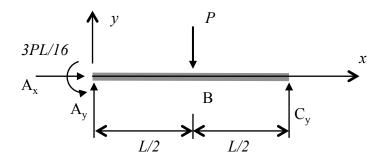
Substituting released problem deflections and flexibility coefficient:

$$-\frac{PL^2}{16EI} + M_A \left(-\frac{L}{3EI}\right) += 0$$

$$M_A = \frac{PL^2}{16EI} \left(-\frac{3EI}{L} \right)$$

Solving the system of compatibility equations yields:

$$M_A = -\frac{3PL}{16}$$



We can find the other reactions using equilibrium;

$$+\rightarrow \sum F_x = 0$$

$$A_x = 0$$

$$\sum M_A=0$$

+ Counterclockwise moments about point A are positive

$$\frac{3PL}{16} - P\left(\frac{L}{2}\right) + C_y(L) = 0$$

 $C_y = \frac{5}{16}P$ (C_y is positive so it acts upward as assumed)

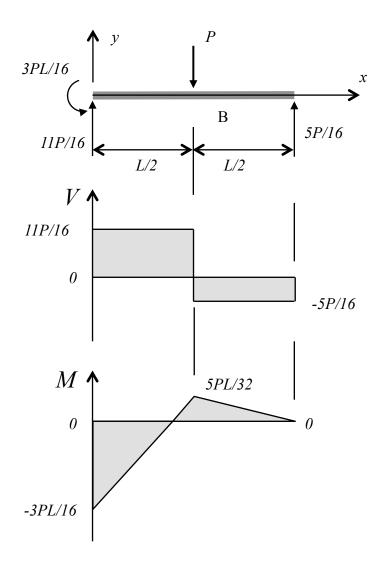
$$+\uparrow \sum F_y = 0$$

$$A_y + C_y - P = 0$$

$$A_y + \frac{5}{16}P - P = 0$$

$$A_y = \frac{11}{16}P$$
 (A_y is positive so it acts upward as assumed)

Shear and bending moment diagrams for the indeterminate beam.



We can find the maximum positive moment at point B using the area under the shear diagram

$$M_B - M_A = \left(\frac{11P}{16}\right) \left(\frac{L}{2}\right)$$

$$M_B = -\frac{3PL}{16} + \frac{11PL}{32} = \frac{5PL}{32}$$